

HW9 Solutions

Ladybugs on a Rotating Disk

Part A. Answer: the same as the angular speed of ladybug 2

Part B. Answer: 2

Hint . Relation between linear and angular speeds

The relation between the linear speed v and angular speed ω of an object is given by

$$V = r\omega,$$

where r is the distance between the object and the axis of rotation.

Part C. Answer: $\frac{a_2}{a_1} = 2$

Hint. Radial (centripetal) acceleration of an object moving on a circle

The magnitude of the radial (centripetal) acceleration of an object moving on a circle is called the centripetal acceleration. It is given by

$$a_c = r\omega^2 = \frac{v_t^2}{r},$$

where ω is the angular velocity of the object, v_t is its tangential velocity, and r is the distance from the axis of rotation.

Part D. Answer: +z

Part E. Answer: -y

9.4.IDENTIFY: $\alpha_z = d\omega_z/dt$. $\alpha_{av-z} = \frac{\Delta\omega_z}{\Delta t}$.

SET UP: $\frac{d}{dt}(t^2) = 2t$

EXECUTE: (a) $\alpha_z(t) = \frac{d\omega_z}{dt} = -2\beta t = (-1.60 \text{ rad/s}^3)t$.

(b) $\alpha_z(3.0 \text{ s}) = (-1.60 \text{ rad/s}^3)(3.0 \text{ s}) = -4.80 \text{ rad/s}^2$.

$$\alpha_{av-z} = \frac{\omega_z(3.0 \text{ s}) - \omega_z(0)}{3.0 \text{ s}} = \frac{-2.20 \text{ rad/s} - 5.00 \text{ rad/s}}{3.0 \text{ s}} = -2.40 \text{ rad/s}^2,$$

which is half as large (in magnitude) as the acceleration at $t = 3.0 \text{ s}$.

EVALUATE: $\alpha_z(t)$ increases linearly with time, so $\alpha_{\text{av-z}} = \frac{\alpha_z(0) + \alpha_z(3.0 \text{ s})}{2}$.
 $\alpha_z(0) = 0$.

9.15. IDENTIFY: Apply constant angular acceleration equations.

SET UP: Let the direction the flywheel is rotating be positive.

$$\theta - \theta_0 = 200 \text{ rev}, \omega_{0z} = 500 \text{ rev/min} = 8.333 \text{ rev/s}, t = 30.0 \text{ s}.$$

EXECUTE: (a) $\theta - \theta_0 = \left(\frac{\omega_{0z} + \omega_z}{2}\right)t$ gives $\omega_z = 5.00 \text{ rev/s} = 300 \text{ rpm}$

(b) Use the information in part (a) to find α_z : $\omega_z = \omega_{0z} + \alpha_z t$ gives

$\alpha_z = -0.1111 \text{ rev/s}^2$. Then $\omega_z = 0$, $\alpha_z = -0.1111 \text{ rev/s}^2$, $\omega_{0z} = 8.333 \text{ rev/s}$ in $\omega_z = \omega_{0z} + \alpha_z t$ gives
 $t = 75.0 \text{ s}$ and $\theta - \theta_0 = \left(\frac{\omega_{0z} + \omega_z}{2}\right)t$ gives $\theta - \theta_0 = 312 \text{ rev}$.

EVALUATE: The mass and diameter of the flywheel are not used in the calculation.

9.24. IDENTIFY: Apply constant angular acceleration equations. $v = r\omega$. A point on the rim has both tangential and radial components of acceleration.

SET UP: $a_{\text{tan}} = r\alpha$ and $a_{\text{rad}} = r\omega^2$.

EXECUTE: (a) $\omega_z = \omega_{0z} + \alpha_z t = 0.250 \text{ rev/s} + (0.900 \text{ rev/s}^2)(0.200 \text{ s}) = 0.430 \text{ rev/s}$

(Note that since ω_{0z} and α_z are given in terms of revolutions, it's not necessary to convert to radians).

(b) $\omega_{\text{av-z}} \Delta t = (0.340 \text{ rev/s})(0.2 \text{ s}) = 0.068 \text{ rev}$.

(c) Here, the conversion to radians must be made to use $v = r\omega$, and

$$v = r\omega = \left(\frac{0.750 \text{ m}}{2}\right)(0.430 \text{ rev/s})(2\pi \text{ rad/rev}) = 1.01 \text{ m/s}.$$

(d) Combining $a_{\text{rad}} = r\omega^2$ and $a_{\text{tan}} = R\alpha$, $a = \sqrt{a_{\text{rad}}^2 + a_{\text{tan}}^2} = \sqrt{(\omega^2 r)^2 + (\alpha r)^2}$.

$$a = \sqrt{\left[((0.430 \text{ rev/s})(2\pi \text{ rad/rev}))^2 (0.375 \text{ m}) \right]^2 + \left[(0.900 \text{ rev/s}^2)(2\pi \text{ rad/rev})(0.375 \text{ m}) \right]^2}.$$

$$a = 3.46 \text{ m/s}^2.$$

EVALUATE: If the angular acceleration is constant, a_{tan} is constant but a_{rad} increases as ω increases.

9.30. IDENTIFY: Treat each block as a point mass, so for each block $I = mr^2$, where r is the distance of the block from the axis. The total I for the object is the sum of the I for each of its pieces.

SET UP: In part (a) two blocks are a distance $L/2$ from the axis and the third block is on the axis. In part (b) two blocks are a distance $L/4$ from the axis and one is a distance $3L/4$ from the axis.

EXECUTE: (a) $I = 2m(L/2)^2 = \frac{1}{2}mL^2$.

(b) $I = 2m(L/4)^2 + m(3L/4)^2 = \frac{1}{16}mL^2(2+9) = \frac{11}{16}mL^2$.

EVALUATE: For the same object I is in general different for different axes.

9.34. IDENTIFY: $K = \frac{1}{2}I\omega^2$. Use Table 9.2 to calculate I .

SET UP: $I = \frac{1}{12}ML^2$. 1 rpm = 0.1047 rad/s

EXECUTE: (a) $I = \frac{1}{12}(117 \text{ kg})(2.08 \text{ m})^2 = 42.2 \text{ kg} \cdot \text{m}^2$.

$\omega = (2400 \text{ rev/min})\left(\frac{0.1047 \text{ rad/s}}{1 \text{ rev/min}}\right) = 251 \text{ rad/s}$. $K = \frac{1}{2}I\omega^2 = \frac{1}{2}(42.2 \text{ kg} \cdot \text{m}^2)(251 \text{ rad/s})^2 = 1.33 \times 10^6 \text{ J}$.

(b) $K_1 = \frac{1}{12}M_1L_1^2\omega_1^2$, $K_2 = \frac{1}{12}M_2L_2^2\omega_2^2$. $L_1 = L_2$ and $K_1 = K_2$, so $M_1\omega_1^2 = M_2\omega_2^2$.

$\omega_2 = \omega_1\sqrt{\frac{M_1}{M_2}} = (2400 \text{ rpm})\sqrt{\frac{M_1}{0.750M_1}} = 2770 \text{ rpm}$

EVALUATE: The rotational kinetic energy is proportional to the square of the angular speed and directly proportional to the mass of the object.

9.45. IDENTIFY: With constant acceleration, we can use kinematics to find the speed of the falling object. Then we can apply the work-energy expression to the entire system and find the moment of inertia of the wheel. Finally, using its radius we can find its mass, the target variable.

SET UP: With constant acceleration, $y - y_0 = \left(\frac{v_{0y} + v_y}{2}\right)t$. The angular velocity

of the wheel is related to the linear velocity of the falling mass by $\omega_z = \frac{v_y}{R}$. The

work-energy theorem is $K_1 + U_1 + W_{\text{other}} = K_2 + U_2$, and the moment of inertia of a uniform disk is $I = \frac{1}{2}MR^2$.

EXECUTE: Find v_y , the velocity of the block after it has descended 3.00 m.

$y - y_0 = \left(\frac{v_{0y} + v_y}{2}\right)t$ gives $v_y = \frac{2(y - y_0)}{t} = \frac{2(3.00 \text{ m})}{2.00 \text{ s}} = 3.00 \text{ m/s}$. For the wheel,

$\omega_z = \frac{v_y}{R} = \frac{3.00 \text{ m/s}}{0.280 \text{ m}} = 10.71 \text{ rad/s}$. Apply the work-energy expression:

$K_1 + U_1 + W_{\text{other}} = K_2 + U_2$, giving $mg(3.00 \text{ m}) = \frac{1}{2}mv^2 + \frac{1}{2}I\omega^2$. Solving for I gives

$$I = \frac{2}{\omega^2} \left[mg(3.00 \text{ m}) - \frac{1}{2}mv^2 \right]. \quad I = \frac{2}{(10.71 \text{ rad/s})^2} \left[(4.20 \text{ kg})(9.8 \text{ m/s}^2)(3.00 \text{ m}) - \frac{1}{2}(4.20 \text{ kg})(3.00 \text{ m/s})^2 \right].$$

$$I = 1.824 \text{ kg} \cdot \text{m}^2. \text{ For a solid disk, } I = \frac{1}{2}MR^2 \text{ gives } M = \frac{2I}{R^2} = \frac{2(1.824 \text{ kg} \cdot \text{m}^2)}{(0.280 \text{ m})^2} = 46.5 \text{ kg}.$$

EVALUATE: The gravitational potential of the falling object is converted into the kinetic energy of that object and the rotational kinetic energy of the wheel.

9.52. IDENTIFY: Use the equations in Table 9.2. I for the rod is the sum of I for each segment. The parallel-axis theorem says $I_p = I_{\text{cm}} + Md^2$.

SET UP: The bent rod and axes a and b are shown in Figure 9.52. Each segment has length $L/2$ and mass $M/2$.

EXECUTE: (a) For each segment the moment of inertia is for a rod with mass $M/2$, length $L/2$ and the axis through one end. For one segment,

$$I_s = \frac{1}{3} \left(\frac{M}{2} \right) \left(\frac{L}{2} \right)^2 = \frac{1}{24} ML^2. \text{ For the rod, } I_a = 2I_s = \frac{1}{12} ML^2.$$

(b) The center of mass of each segment is at the center of the segment, a distance of $L/4$ from each end. For each segment, $I_{\text{cm}} = \frac{1}{12} \left(\frac{M}{2} \right) \left(\frac{L}{2} \right)^2 = \frac{1}{96} ML^2$. Axis b is a distance $L/4$ from the cm of each segment, so for each segment the parallel axis theorem gives I for axis b to be $I_s = \frac{1}{96} ML^2 + \frac{M}{2} \left(\frac{L}{4} \right)^2 = \frac{1}{24} ML^2$ and

$$I_b = 2I_s = \frac{1}{12} ML^2.$$

EVALUATE: I for these two axes are the same.

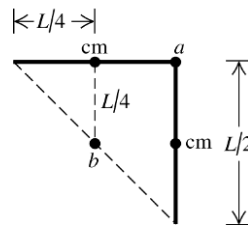


Figure 9.52

9.55. IDENTIFY: Apply $I = \int r^2 dm$ and $M = \int dm$.

SET UP: For this case, $dm = \gamma x dx$.

EXECUTE: (a) $M = \int dm = \int_0^L \gamma x dx = \gamma \frac{x^2}{2} \Big|_0^L = \frac{\gamma L^2}{2}$

(b) $I = \int_0^L x^2 (\gamma x) dx = \gamma \frac{x^4}{4} \Big|_0^L = \frac{\gamma L^4}{4} = \frac{M}{2} L^2$. This is larger than the moment of inertia of a uniform rod of the same mass and length, since the mass density is greater farther away from the axis than nearer the axis.

(c) $I = \int_0^L (L-x)^2 \gamma x dx = \gamma \int_0^L (L^2 x - 2Lx^2 + x^3) dx = \gamma \left(L^2 \frac{x^2}{2} - 2L \frac{x^3}{3} + \frac{x^4}{4} \right) \Big|_0^L = \gamma \frac{L^4}{12} = \frac{M}{6} L^2$.

This is a third of the result of part (b), reflecting the fact that more of the mass is concentrated at the right end.

EVALUATE: For a uniform rod with an axis at one end, $I = \frac{1}{3} ML^2$. The result in (b) is larger than this and the result in (c) is smaller than this.

9.75. IDENTIFY: Apply conservation of energy to the system consisting of blocks A and B and the pulley.

SET UP: The system at points 1 and 2 of its motion is sketched in Figure 9.75.

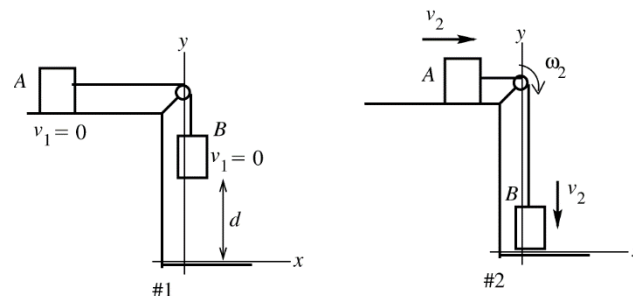


Figure 9.75

Use the work-energy relation $K_1 + U_1 + W_{\text{other}} = K_2 + U_2$. Use coordinates where $+y$ is upward and where the origin is at the position of block B after it has descended. The tension in the rope does positive work on block A and negative work of the same magnitude on block B, so the net work done by the tension in the rope is zero. Both blocks have the same speed.

EXECUTE: Gravity does work on block B and kinetic friction does work on block A. Therefore $W_{\text{other}} = W_f = -\mu_k m_A g d$.

$K_1 = 0$ (system is released from rest)

$U_1 = m_B g y_{B1} = m_B g d$; $U_2 = m_B g y_{B2} = 0$

$$K_2 = \frac{1}{2}m_A v_2^2 + \frac{1}{2}m_B v_2^2 + \frac{1}{2}I\omega_2^2.$$

But $v(\text{blocks}) = R\omega(\text{pulley})$, so $\omega_2 = v_2/R$ and

$$K_2 = \frac{1}{2}(m_A + m_B)v_2^2 + \frac{1}{2}I(v_2/R)^2 = \frac{1}{2}(m_A + m_B + I/R^2)v_2^2$$

Putting all this into the work-energy relation gives

$$m_B g d - \mu_k m_A g d = \frac{1}{2}(m_A + m_B + I/R^2)v_2^2$$

$$(m_A + m_B + I/R^2)v_2^2 = 2gd(m_B - \mu_k m_A)$$

$$v_2 = \sqrt{\frac{2gd(m_B - \mu_k m_A)}{m_A + m_B + I/R^2}}$$

EVALUATE: If $m_B \gg m_A$ and I/R^2 , then $v_2 = \sqrt{2gd}$; block B falls freely. If I is very large, v_2 is very small. Must have $m_B > \mu_k m_A$ for motion, so the weight of B will be larger than the friction force on A . I/R^2 has units of mass and is in a sense the “effective mass” of the pulley.

9.8.IDENTIFY: $\alpha_z = \frac{d\omega_z}{dt}$. $\theta - \theta_0 = \omega_{\text{av-z}}t$. When ω_z is linear in t , $\omega_{\text{av-z}}$ for the time interval

$$t_1 \text{ to } t_2 \text{ is } \omega_{\text{av-z}} = \frac{\omega_{z1} + \omega_{z2}}{t_2 - t_1}.$$

SET UP: From the information given, $\alpha_z = \frac{\Delta\omega}{\Delta t} = \frac{4.00 \text{ rad/s} - (-6.00 \text{ rad/s})}{7.00 \text{ s}} = 1.429 \text{ rad/s}^2$.

$$\omega_z(t) = -6.00 \text{ rad/s} + (1.429 \text{ rad/s}^2)t.$$

EXECUTE: (a) The angular acceleration is positive, since the angular velocity increases steadily from a negative value to a positive value.

(b) It takes time $t = -\frac{\omega_{0z}}{\alpha_z} = -(-6.00 \text{ rad/s})/(1.429 \text{ rad/s}^2) = 4.20 \text{ s}$ for the

wheel to stop ($\omega_z = 0$). During this time its speed is decreasing. For the next 2.80 s its speed is increasing from 0 rad/s to +4.00 rad/s.

(c) The average angular velocity is $\frac{-6.00 \text{ rad/s} + 4.00 \text{ rad/s}}{2} = -1.00 \text{ rad/s}$. $\theta - \theta_0 = \omega_{\text{av-z}}t$

then leads to displacement of -7.00 rad after 7.00 s.

EVALUATE: When α_z and ω_z have the same sign, the angular speed is increasing; this is the case for $t = 4.20 \text{ s}$ to $t = 7.00 \text{ s}$. When α_z and ω_z have opposite signs, the angular speed is decreasing; this is the case between $t = 0$ and $t = 4.20 \text{ s}$.